

1 **CHARACTERIZATION AND OPTIMIZATION OF HYBRID**
2 **CARBON-GLASS EPOXY COMPOSITES UNDER COMBINED**
3 **LOADING**
4

5 **V. Infante^a, J.F. Aguilar Madeira^{a,b}, Rui B. Ruben^{c,1}, F. Moleiro^a,**
6 **Sofia Teixeira de Freitas^d**
7

8 ^aLAETA, IDMEC, Instituto Superior Técnico, Universidade de Lisboa, Av. Rovisco Pais 1, 1049-001
9 Lisboa, Portugal.

10 ^b Department of Mathematics, ISEL, IPL, Rua Conselheiro Emídio Navarro, 1949-014 Lisboa, Portugal

11 ^c CDRsp - ESTG - Polytechnic Institute of Leiria, Campus 2 Morro do Lena – Alto do Vieiro, 2411-901
12 Leiria, Portugal.

13 E-mail: rui.ruben@ipleiria.pt

14 ^d Faculty of Aerospace Engineering, Delft Univ. of Technology, Kluyverweg 1, Delft, Netherlands.
15
16

17 **ABSTRACT**

18 This work is intended to characterize the mechanical behavior of hybrid carbon-glass composites plates
19 under combined loading of bending and torsion, and to determine the optimal ply fiber orientations to
20 minimize the maximum out of plane displacement under such loading conditions. Hybrid composite
21 plates were manufactured with 10 plies each and different stacking sequences using hand lay-up, with
22 carbon fiber and glass fiber reinforcements in an epoxy matrix. Two experimental setups (involving two
23 distinct boundary conditions) are here considered to test the composite plates, both simulating combine
24 loading of bending and torsion. Numerical simulations of the experimental tests were performed in
25 ABAQUS[®] and validated with the experimental data. Using the ply fiber orientations as design variables,
26 the hybrid composite plates were then optimized using GLODS – Global and Local Optimization using
27 Direct Search. The objective function of minimization the maximum out of plane displacement is carried
28 out through an interactive cycle between GLODS and ABAQUS[®]. Specimens of three optimized
29 laminates were also manufactured for experimental validation. The optimization process contributed to
30 improve the performance of the hybrid composites plates in more than 30% when compared to some non-
31 optimized plates.
32
33
34

¹ Corresponding autor. Address: ESTG - Polytechnic Institute of Leiria, Campus 2 Morro do Lena – Alto do Vieiro, 2411-901 Leiria, Portugal. Tel: +351 244830010. Email: rui.ruben@ipleiria.pt

35 **KEYWORDS:**

36A. Hybrid composites

37B. Carbon and Glass fibers

38C. Finite element analysis

39D. Optimization

40

41. **INTRODUCTION**

42 Polymer matrix composites are becoming increasingly important in high technology applications,
43 particularly in the aeronautical industry. Aircraft designers typically look for high specific stiffness, high
44 specific strength, low density and reduced cost while selecting the material systems for different structural
45 components. However, due to their characteristic high stiffness and strength, most of the composite
46 materials, using single material reinforcements, have limited toughness values.

47 Hybrid composites may offer an effective solution for increasing the toughness of a composite material.
48 By combining two fiber types in a single composite, one can take advantage of both fiber properties and
49 try to minimize some of their disadvantages. The hybrid composites concept consist in combining fibers
50 with high modulus and high specific strength, relatively brittle and generally expensive, with fibers with
51 low modulus and lower specific strength, more ductile and frequently cheaper. Carbon fiber laminates are
52 widely used in several applications due to their high specific modulus and strength. Nonetheless, the
53 impact strength of these composites is generally lower than many metal alloys. Previous research has
54 shown that an efficient method of increasing the tensile ultimate strain [1], impact properties [2] as well
55 as flexural strength properties [3, 4] of carbon fiber composites is to add some percentage of low modulus
56 fibers, such as glass fibers, both using intra- or inter- ply hybridization. With this hybridization, the
57 carbon fiber can still provide the necessary stiffness, while the final composite material is cheaper and
58 more damage tolerant due to the presence of the glass fibers. The final mechanical properties of the
59 hybrid composite can be tailored by changing the stacking sequence of the plies (interply hybrid
60 composites – different fiber materials along the stacking sequence) and the volume fraction of each fiber
61 (intraply hybrid composite – different fiber material inside the plies) [5].

62 The impact performance of interply hybrid composites has been extensively reported by several
63 works found in literature which concluded that hybrid carbon-glass epoxy composites show an

64 improvement in damage tolerance and carrying capacity after impact in comparison with carbon/epoxy
65 laminates with slight reduction of stiffness [2, 6–8]. Gonzalez et al. (2014) [9] further demonstrates that
66 the stacking sequence of the carbon-glass hybrid composites has a significant influence on the failure
67 mechanisms sequence, growth and interactions during impact tests and compression after impact tests.

68 Under bending conditions, literature suggests that the best flexural properties of hybrid glass-carbon
69 laminates are obtained when the carbon plies are positioned at the exterior [10]. The same study
70 concludes that the tensile strength was insensitive to the relative position of the glass-carbon plies and an
71 alternate glass-carbon stacking sequence favours compressive strength. Dong et al. [3, 4] tested hybrid
72 laminates under three point bending and observed that the highest flexural strength is achieved when
73 specimens contain 24% of glass fibers. These hybrid laminates show an increase of 8% and 3% when
74 compared to the ones with full carbon and glass configurations, respectively. Simulation studies show
75 much higher increases in flexural strength than the corresponding experiments. When looking to intraply
76 hybrid laminates, literature shows that the fiber volume fraction has a significant influence on the flexural
77 strength. Dong et al. [11] claims that in general in order to improve flexural strength, the fiber volume
78 fraction of glass/epoxy plies needs to be higher than that of carbon/epoxy plies [12]. Few studies were
79 found in which an optimization of the stacking sequence was performed. Kalantari et al. [13, 14] run
80 multiobjective optimization analysis with several objectives such as flexural strength, weight and costs.
81 The results suggest that the hybridization of carbon fiber reinforced composites with glass fibers, and vice
82 versa, not only improves the flexural strength but can also optimize the weight and cost of the composite
83 structure.

84 Hybrid laminates can also be used for health monitoring purposes, for example. Loading the hybrid
85 composite in the fiber direction in tension, will cause the more brittle fibers to fail before the more ductile
86 fibers. Also, in a laminate with only carbon fiber reinforcements, the replacement of the middle plies by
87 cheaper glass fibers can significantly reduce its cost, while the flexural properties remain almost
88 unaffected [15]. The extensive work performed on characterizing the properties of fiber hybridization
89 under tensile, flexural and impact loading is reported in the comprehensive review of Swolfs et al. [15].

90 Optimization of composite materials in the field of design of ply orientation has drawn extensive
91 attention over the past two decades [16]. In literature, for example, Todoroki et al. [17] use the Genetic
92 Algorithm (GA) to optimize ply orientation by considering the enforcement constraints to limit the

93 number of continuous orientation plies. Toropov et al. [18] applied GA to optimize the weight and
94 manufacturability of composite aircraft components. Zhu et al. [19] applied GA to a multiobjective linear
95 variable weights to optimize the stacking sequence and ply thickness of carbon fiber composite struts.
96 The advantage of using such heuristic algorithms is that there is no need to derive design sensitivity
97 information and also the possibility of converging to a global optimum. Another class of methods that
98 does not need any gradient information, but rather require only the value of the objective function, are the
99 direct search methods [20]. The Direct MultiSearch (DMS) [21] is a direct search method for
100 multiobjective optimization. In fact, it has been recently used by Araujo et al. [22] to find the optimal
101 positioning of surface bonded sensors and actuators for damping maximization. It has also been used to
102 solve the problem of minimum weight and maximum damping of viscoelastic sandwich plates by
103 Madeira et al. [23, 24] and to the design optimization of laminated composite plates with piezoelectric
104 layers by Franco Correia et al. [25], among other applications. Global and Local Optimization using
105 Direct Search (GLODS) is a direct search method designed to deal with global optimization problems
106 with a single objective function. This method has been used previously by Monte et al. [26] to optimize
107 the carbon fibers orientation of composite plates in order to minimize their global out of plane
108 displacement. The optimization process has been validated with experimental results. In this paper
109 GLODS is used to optimize a hybrid composite plate under combined loading of bending and torsion.

110 The works found in literature up to now on hybrid composites, focus on their characterization and
111 optimization under one single loading, such as isolated tensile, or isolated compression or isolated
112 bending. Nevertheless, in real composite structures such loading conditions are rare and a combine
113 loading is often more realistic. For instance, the combined loading of bending and torsion that often
114 occurs in wing skins of an aircraft. Therefore, it is considered essential for their applicability, to
115 investigate the properties of hybrid composites under combined loading of bending and torsion. This
116 paper aims exactly to fill in this gap of knowledge in the current state of the art and go one step further by
117 optimizing the hybrid composite stacking sequence for combined loading conditions.

118 The main objective of this study is therefore to characterize the mechanical behavior of hybrid
119 composites plates under combined loading of bending and torsion, as well as, to determine the optimal
120 ply fiber orientations to minimize the maximum out of plane displacement under this loading condition.

121

1222. **MATERIALS AND SPECIMENS**

123 The two fiber types used in hybrid composites are typically referred to as low elongation (LE) and
 124 high elongation (HE) fibers [15]. In the present work, the LE and HE fibers are HS carbon fibers and E-
 125 glass fibers, respectively, and these can be combined in many different configurations. The different non-
 126 optimized specimens here analysed were produced using the most common ply fiber orientations of 0°
 127 and 90° with different stacking sequences.

128 The specimens were produced using the T700S carbon and E-glass reinforcements in an epoxy
 129 matrix resin. The matrix properties are presented in Table 1, in agreement with the supplier (Sicomín®)
 130 Technical Datasheet. Specifications for the fabrics and filaments used as reinforcements are presented in
 131 Table 2. And in Table 3, the fiber volume fraction of hybrid composites and ply and laminate thickness
 132 were presented.

133 **Table 1.** Matrix components and properties.

<i>Epoxy resin SR 1500 + Curing agent SD 2503</i>	
Young's Modulus [GPa]	2.86
Tensile Strength [MPa]	71
Density [g/cm ³]	1.00
Poisson's Ratio	0.3

134

135 **Table 2.** Mechanical and acoustic properties of the specimens.

Property	HS Carbon	E-Glass
Type of fabric	T700S	12.600
Fabric aerial weight [g/m ²]	225	600
Filament diameter [µm]	7	17
Filament maximum elongation	2.1%	4.8%
Filament Young's modulus [GPa]	230	74
Filament tensile strength [MPa]	4900	2500
Filament density [g/cm ³]	1.80	2.54
Filament Poisson's ratio	0.25	0.3

136

137 **Table 3.** Fiber volume fraction of hybrid composites and ply and laminate thickness.

Parameter	Values
Thickness of a carbon layer	0.24 mm
Thickness of a glass layer	0.43 mm
Total specimen thickness	3.16 mm
Carbon density	0.5071
Glass fraction	0.5860

138

139 Five hybrid composite plates were produced with unidirectional dry fibers using the hand lay-up
 140 (HLU) technique. The mixing ratio by weight of the resin and the fibers was 100:33, as recommended by
 141 the resin manufacturer.

142 The curing process consisted in two phases. Initially, each specimen was placed inside a vacuum
 143 bag with breather assemblies. Air was extracted from the material using a vacuum pump, and cure
 144 occurred at a controlled room temperature of 20 °C. This vacuum bagging process consolidates the plies
 145 and significantly reduces voids in the composite material. After 24 hours in room temperature, each
 146 specimen went through a secondary post-cure, during which the specimen is kept inside a furnace for 8
 147 hours at 60 °C. Five specimens with 10 plies were produced. The number of plies and fiber orientations of
 148 each specimen, P1 to P5, are presented in Table 4.

149 **Table 4.** Material stacking sequences and ply fiber orientations of specimens made with carbon fiber (C) and glass fiber (G)
 150 reinforcements.

Specimen	Number of plies	Material stacking sequence	Ply fiber orientations
P1			$[0_3/90_2]_s$
P2			$[0_2/90_2/0]_s$
P3	10	$[C_3/G_2]_s$	$[0_4/90]_s$
P4			$[0_2/90_3]_s$
P5			$[0/90/0/90_2]_s$

151

152 These plies of either carbon fiber or glass fiber reinforced polymer composites may be considered
 153 transversely isotropic materials, as a result of their microscopic heterogeneity. It is therefore necessary to
 154 identify at least five independent elastic properties to characterize the behavior of the homogenized
 155 composite plies [27]. One of the main difficulties encountered in the computation simulation of the hybrid
 156 composites is the determination of these properties for each specimen ply materials. To calculate the
 157 elastic properties it was first necessary to find the fiber volume ratio for each of them. Hence, two
 158 additional specimens were produced, one with only carbon fiber and another with only glass fiber plies
 159 both orientated at 0°. The total mass of the plies was measured before starting the hand-lay-up and both
 160 composites were weighted after the post-cure process to calculate the fiber volume ratio for the carbon
 161 fiber and for the glass fiber composites. The fiber volume ratios obtained for the carbon fiber and glass
 162 fiber were the following: $v_{cf}= 0.5071$ and $v_{gf}= 0.5860$.

163 With the fiber volume ratios for each material and the data of Table 1 and Table 2, it was finally
 164 possible to calculate the elastic properties for each specimen material layer, with the numerical method
 165 proposed and developed by J. M. Guedes et al. [28]: PREMAT. This finite element model is based on a
 166 homogenization theory, which assumes that the composite material is locally formed by the spatial
 167 repetition of very small microstructures called Representative Elementary Volumes or RVE.
 168 Theoretically, it is mechanically admissible to use the elastic properties given by the periodic

169 homogenization theory of this model, due to the relative dimensions of the filament diameter compared to
 170 the thickness of each ply [29]. The elastic properties, obtained using this method, are presented in Table
 171 5. The material strength of each ply used in the failure analysis was also obtained by the homogenization
 172 theory [28] (table 5).

173
 174

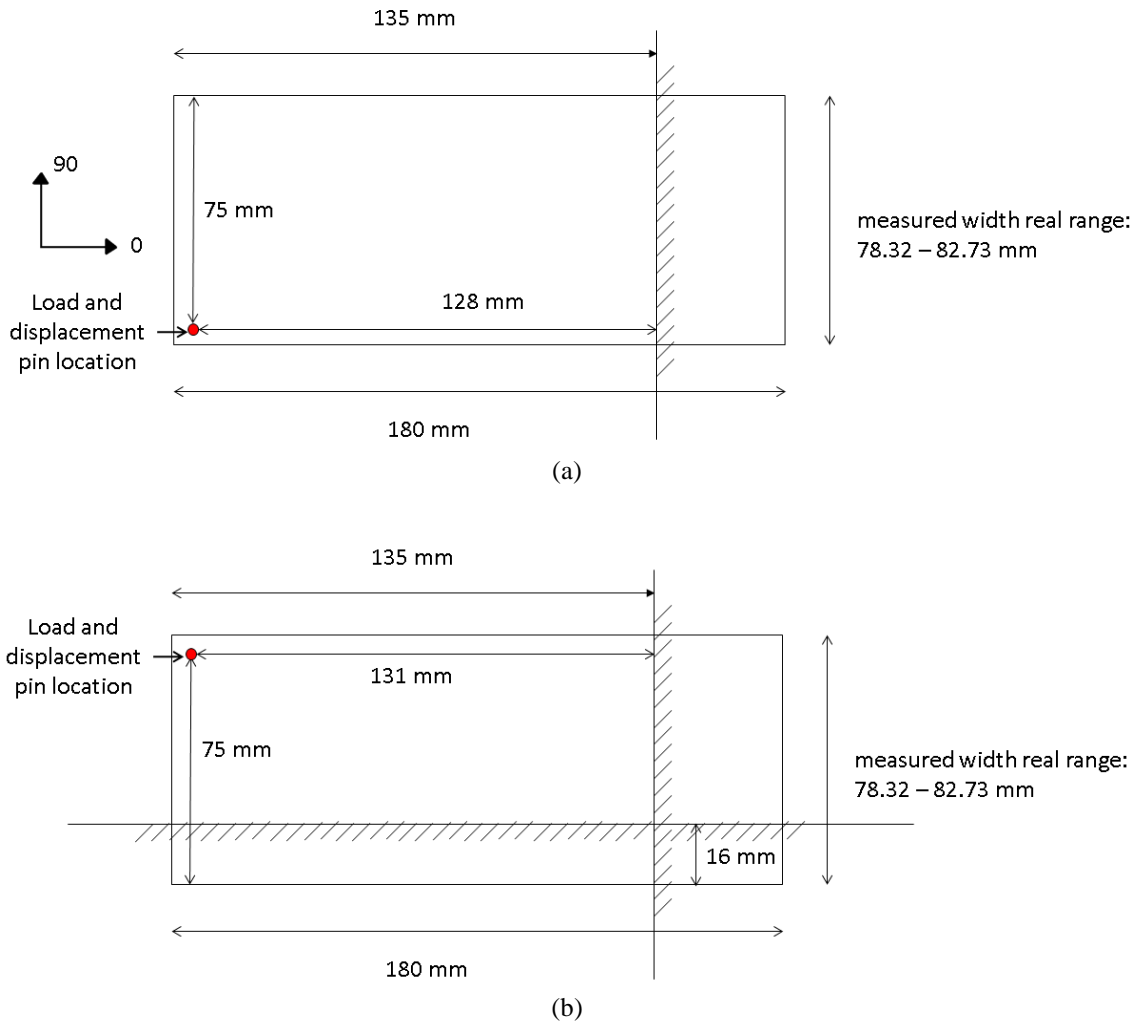
Table 5. Properties of ply materials.

Ply material property			Carbon Fiber	Glass fiber
Longitudinal tensile modulus	E11		117.84 GPa	44.49 GPa
Transverse tensile modulus	E22		9.73 GPa	11.34 GPa
In-plane Poisson ratio	ν_{12}		0.27	0.30
Transverse Poisson ratio	ν_{23}		0.25	0.23
In-plane shear modulus	$G_{12}=G_{13}$		3.34 GPa	3.89 GPa
Transverse Shear modulus	G_{23}		2.40 GPa	2.78 GPa
Longitudinal tensile strength	X_T		2231 MPa	1100 MPa
Longitudinal Compressive strength	X_C		1082 MPa	675 MPa
Transverse tensile strength	Y_T		29 MPa	35 MPa
Transverse compressive strength	Y_C		100 MPa	120 MPa
In-plane shear strength	$S_{12}=S_{13}$		60 MPa	80 MPa
Transverse shear strength	S_{23}		32 MPa	46 MPa

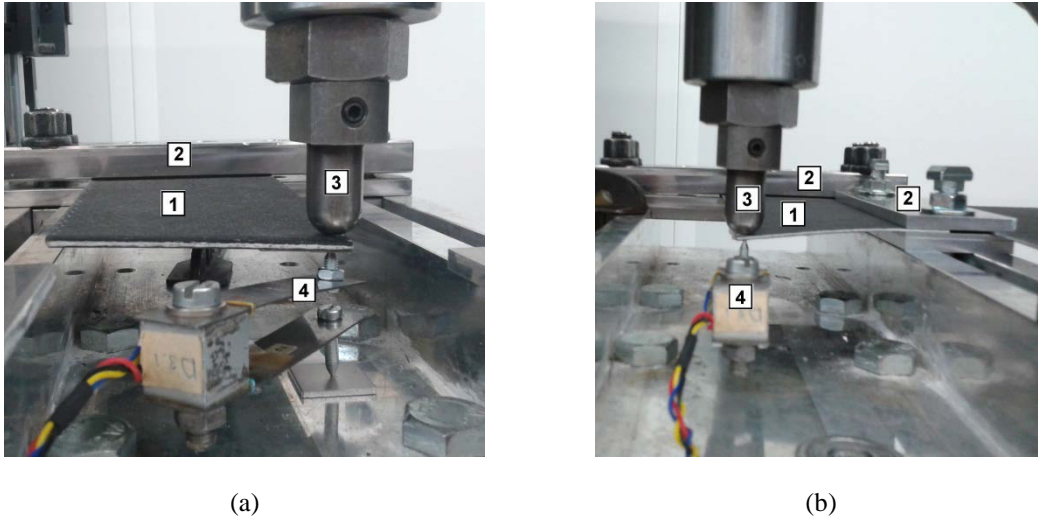
175

1763. **EXPERIMENTAL METHODS**

177 Two types of experimental tests were performed, and deformations were measured for each specimen,
178 subjected to bending-torsion loading. The constraints and the loading location in each experimental setup
179 test are illustrated in Figure 1.
180



181 **Figure 1.** Illustration of the conducted experimental setup tests: (a) plate fixed in one side, (b) plate fixed in two sides.
182 The experimental tests were conducted in Delft Aerospace Structures and Materials Laboratory
183 (DASML) at Delft University of Technology. Loading occurs at a rate of 2 mm/min and out of plane
184 displacements are registered using a mechanical clip gauge in the vertical direction of the load location. A
185 picture of each experimental setup is shown in Figure 2.
186



187 **Figure 2.** Experimental bending-torsion tests with the hybrid composite plates: (a) one side fixed; (b) two sides fixed.
 188 1 – Plate; 2 – Fixed side; 3 – Load application; 4 – Mechanical clip gauge.
 189

190 **3.1 Experimental Results**

191 The experimental results are presented in Table 6. The out of plane displacement values were taken at
 192 the pin located in the corner of the specimen (applied load location – see Figure 1) for a loading of 50N.
 193 This load has been selected to ensure that the specimen is kept within the linear elastic range and prior to
 194 any damage initiation. .

195

196

Table 6. Out of plane displacements obtained in experimental tests for different loading values.

Specimen	Material stacking sequence	Ply fiber orientations	Experimental test	Loading [N]	Experimental displacement [mm]
P1	[C ₃ /G ₂] _s	[0 ₃ /90 ₂] _s	One side fixed	50	3.71
P2		[0 ₂ /90 ₂ /0] _s			3.89
P3		[0 ₄ /90] _s			3.49
P4		[0 ₂ /90 ₃] _s			4.40
P5		[0/90/0/90 ₂] _s			4.33
P1	[C ₃ /G ₂] _s	[0 ₃ /90 ₂] _s	Two sides fixed	50	1.69
P2		[0 ₂ /90 ₂ /0] _s			1.37
P3		[0 ₄ /90] _s			2.40
P4		[0 ₂ /90 ₃] _s			1.26
P5		[0/90/0/90 ₂] _s			1.16

197

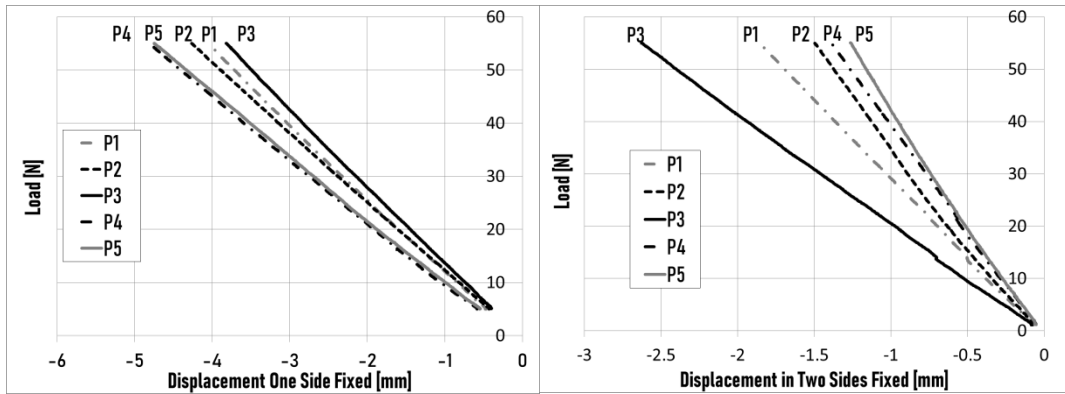


Figure 3. Load-displacement results obtained in the experimental tests.

198
199
200

201 With the coordinate system of Figure 1 and the data in Table 4, the influence of the fiber orientation
202 of the plies is first analyzed, since all specimens have the same material distribution. By comparing the
203 slope of the graphs in Figure 3, it can be observed that the number of plies at 0° and 90° fiber orientations
204 in each specimen is extremely important for the displacement values. It can be easily seen that for the one
205 side fixed experimental tests the most requested fibers are at 0° , whereas in the experimental test with two
206 sides fixed, the most requested fibers are at 90° .

207

208.4. NUMERICAL SIMULATIONS BY FINITE ELEMENT ANALYSIS

209 A parametric finite element model of hybrid composite plates was developed using ABAQUS[®]. With
210 the developed parametric model, it is possible to analyze plates with different plies orientation. This
211 parametric model is very useful for the optimization process. In the finite element analysis (FEA) all five
212 plates presented in Table 4 were numerically simulated, in order to validate the numerical results with the
213 experimental out of plane displacements presented in Table 6. A failure analysis was performed based on
214 the FEA results using Hashin's failure criteria [30, 31] for all plates, in order to ensure that the 50 N load
215 is within the linear elastic domain and prior to any damage initiation.

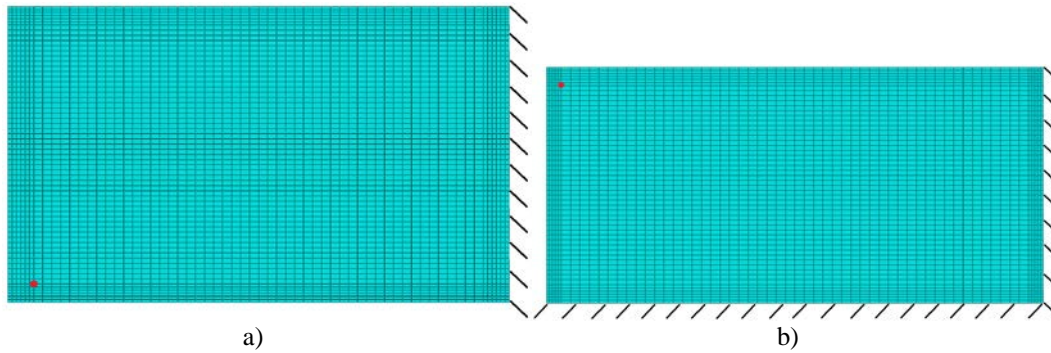
216 Plies elastic properties considered for the laminates are shown in Table 5, which were obtained with
217 the aforementioned homogenization method. For each experimental setup (one side fixed and two sides
218 fixed), two models were built: one using a 4 node linear shell element (S4), and another one using an 8
219 node quadratic shell element (S8). A convergence test was done for all 5 plates for both boundary
220 conditions. The P5 plate with one side fixed has the higher displacement value, so also have the slower
221 convergence curve as is showed in Table 7. So, after the convergence test, a significant number of

222 elements was used to avoid any loss of precision in the optimization method. The same number of
 223 elements was used for S4 and S8 shell elements. Specifically, depending on the experimental setup, 3844
 224 and 3224 elements were used, for one side fixed and two sides fixed, respectively. Computational
 225 displacement was measured at the load point, as depicted in Figure 4.

226 **Table 7.** Convergence test considering P5 with one fixed side.

Number of elements	S4 element		S8 element	
	Computational displacement (mm)	Maximum σ_{11} (MPa)	Computational displacement (mm)	Maximum σ_{11} (MPa)
704	3.93	154.5	3.94	156.8
1344	3.93	154.5	3.94	156.8
1924	3.94	154.5	3.94	156.8
2184	3.94	154.5	3.94	156.8
3244	3.94	154.5	3.94	156.8
3844	3.94	154.5	3.94	156.8
4154	3.94	154.5	3.94	156.8

227
228



229
230

231 **Figure 4.** Mesh, boundary conditions, and load/ computational displacement point in red: a) one side fixed; b) two sides fixed.

232

233 **4.1 FEA Numerical Results**

234 Table 8 presents the out of plane displacements obtained using the FEA as well as the corresponding
 235 experimental values. The first immediate observation is that the displacement obtained from the linear
 236 and quadratic shell elements models are almost equal. Therefore, the linear 4 node shell element proves to
 237 be sufficient for the optimization process while enabling a faster process. Comparing experimental with
 238 numerical results, the discrepancy for the one side fixed plates is smaller than for the two sided fixed
 239 plates. For all cases, the measured displacement in the experiments is larger than the displacement
 240 obtained by FEA. This has also been observed in previous works found in literature, where the flexural
 241 stiffness measured in the experiments was lower than the one predicted by the numerical and analytical
 242 models [3, 4]. The discrepancy is most probably related with the degree of fixation stiffness at the
 243 boundary conditions. In theory, fixed constraints do not allow any displacement or rotation, which is in

244 fact numerically simulated in the model. However, on the actual experimental test, absolute zero
 245 displacements and rotations are very difficult to assure and small displacement and rotations probably
 246 occur in fixed sides. Hence, for two sides fixed plates the difference between experimental and
 247 computational analysis is naturally higher. In order to better replicate the experimental conditions, fixed
 248 constraints that allow infinitesimal rotations were considered (i.e. partially fixed). Boundary conditions
 249 were simulated with standard 8 node elements for steel grips. Contrary to fixed conditions, for which
 250 there is no gap between steel grips and plate (Figure 5a), for partially fixed conditions, a 5 μm gap is used
 251 between top grip and plate, allowing a maximum rotation of 0.1 degrees (Figure 5b). The results of this
 252 adapted model are shown in Table 9. As expected, the discrepancy between the model and experiments is
 253 now smaller, but some differences still persist. This might be related with the ply properties assumed in
 254 the model, which depend on thickness, and the fact that a perfect homogenized ply is almost impossible
 255 to manufacture. Figure 6 shows plots of the out of plane displacement field from the FEAs with totally
 256 and partially fixed rotation.

257 Table 8 also presents the results for Hashin's fiber and matrix failure initiation criterion. For both
 258 fiber and matrix criteria, table 8 only shows the highest value, which is in all cases the tensile one. As is
 259 possible to observe all plates (one sided and two sided fixed) have Hashin's criterion values below 1,
 260 which means that no failure occurs.

261 **Table 8.** Out of plane displacements and discrepancy percentage between the experimental and the computational analysis using the
 262 4 nodes element.

Specimen	Experimental test	Experimental displacement [mm]	Computational displacement (4 nodes) [mm]	Computational displacement (8 nodes) [mm]	Δ %	Hashin's fiber initiation criterion	Hashin's matrix initiation criterion
P1	One side fixed	3.71	3.12	3.12	15.9	0.0028	0.0233
P2		3.89	3.57	3.56	8.2	0.0038	0.0124
P3		3.49	3.04	3.04	12.9	0.0027	0.0343
P4		4.40	3.58	3.58	18.6	0.0039	0.0123
P5		4.33	3.94	3.94	9.0	0.0048	0.0142
P1	Two sides fixed	1.69	1.43	1.43	15.4	0.0004	0.4655
P2		1.37	0.91	0.91	33.3	0.0002	0.1785
P3		2.40	1.70	1.70	29.2	0.0007	0.6909
P4		1.26	0.91	0.90	28.2	0.0002	0.1741
P5		1.16	0.77	0.77	33.6	0.0002	0.1255

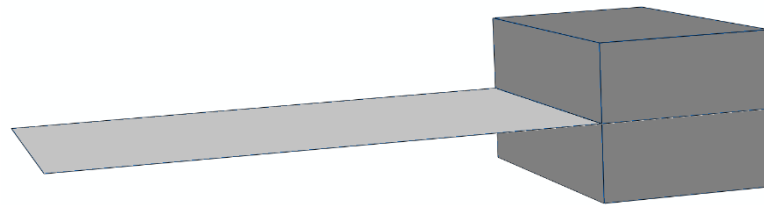
263

264 **Table 9.** Out of plane displacements and discrepancy percentage between the experimental and the computational analysis using
 265 partially fixed rotation (instead of totally fixed).

Specimen	Experimental test	Experimental displacement [mm]	Totally fixed [mm]	Partially fixed [mm]	Δ %
----------	-------------------	--------------------------------	--------------------	----------------------	------------

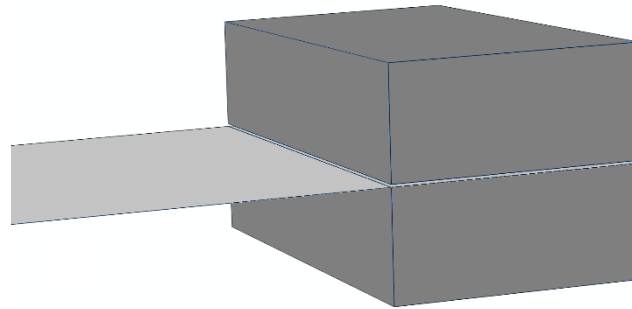
P1		3.71	3.12	3.50	5.7
P2	One side fixed	3.89	3.57	3.95	-1.5
P3		3.49	3.04	3.43	1.7
P4		4.40	3.58	3.97	9.8
P5		4.33	3.94	4.32	0.2
P1		1.69	1.43	1.67	1.2
P2	Two sides fixed	1.37	0.91	1.13	17.5
P3		2.40	1.70	1.96	18.3
P4		1.26	0.91	1.12	11.1
P5		1.16	0.77	0.98	15.7

266



267
268

a)

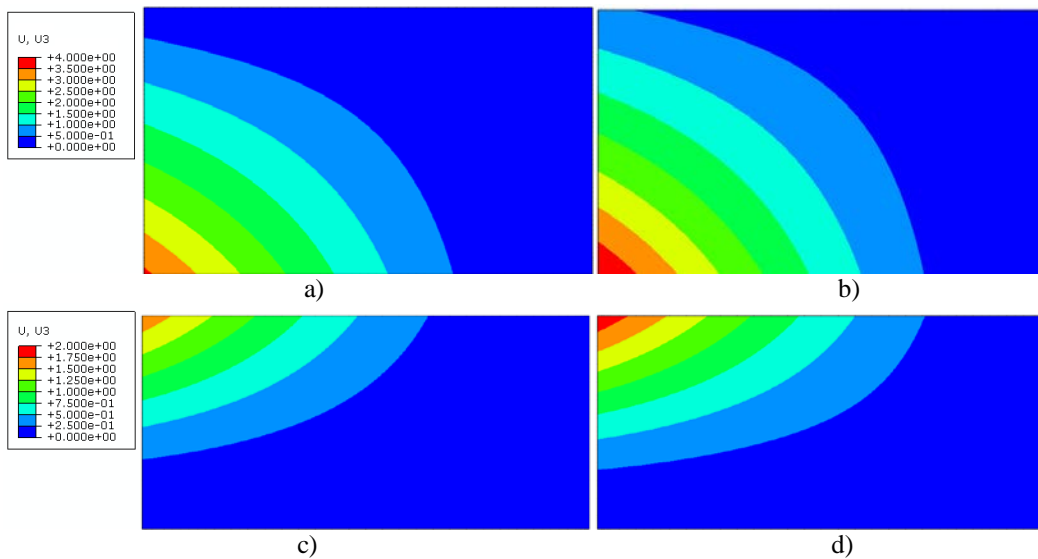


269
270

b)

271
272

Figure 5. a) One side totally fixed and b) One side partially fixed (gap was increased 100 times for visualization purposes).



273
274

275
276

277
278
279

Figure 6. Displacements in mm: a) and b) P1 with one side fixed; c) and d) P1 with two sides fixed; b) and d) with partially fixed rotations.

280 **5. OPTIMIZATION**

281

282 The optimization of the specimens was carried out using GLODS, a single objective global
283 optimization method that uses an efficient multistart strategy. The objective function is focused on the
284 minimization of the maximum out of plane displacement value. The objective function evaluation is made
285 through a finite element analysis (FEA) software ABAQUS®. For clarity, a brief introduction to GLODS
286 is given followed by the optimization method used, in order to introduce the optimization problem solved
287 and present the optimal results.

288

289 **5.1 GLODS (Global and Local Optimization using Direct Search) method**

290 A constrained nonlinear optimization problem can be mathematically formulated as:

291 Find n design variables:

292 $\mathbf{x} = (x_1, x_2, \dots, x_n)^T \in \Omega \subseteq \mathbb{R}^n,$ (1)

293 which minimize:

294 $\min_{\mathbf{x} \in \Omega} f(\mathbf{x})$ (2)

295 where $f: \Omega \subseteq \mathbb{R}^n \rightarrow \mathbb{R} \cup \{+\infty\}$ represents a real-extended value function and $\Omega \subseteq \mathbb{R}^n$ a compact set,
296 defining the problem feasible region.

297 GLODS [32] is a solver suited for global constrained single optimization, which does not use any
298 derivatives of the objective functions.

299 Using direct search of directional type, the method alternates between a search step, where
300 potentially good regions are located, and a poll step where the previously located regions are explored.
301 This exploration is made through the launching of several pattern search methods, one in each of the
302 regions of interest. Differently from a multistart strategy, the several pattern search methods will merge
303 when sufficiently close to each other. The goal of GLODS algorithm is to end with as many active pattern
304 searches as the number of local minimizers, which would allow locating easily the possible global
305 extreme value. Similarly to other derivative-free optimization algorithms, it makes use of the extreme
306 barrier function $f_\Omega(\mathbf{x})$ defined as:

307 $f_\Omega(\mathbf{x}) = \begin{cases} f(\mathbf{x}) & \text{if } \mathbf{x} \in \Omega \\ +\infty & \text{otherwise} \end{cases}$ (3)

308 Unfeasible points will not be evaluated, being the corresponding objective function value set equal to
309 $+\infty$. This approach allows dealing with black-box type constraints, where only a yes/no type of answer is

310 returned. Several details are omitted in the present text and the reader is referred to Custodio et al. [32] for
 311 a more complete description.

312
 313 **5.2 Optimal design formulation**

314 In this work, the objective function is to minimize the maximum out of plane displacement in a
 315 composite specimen made with 10 layers with symmetric material stacking sequence $[C_3/G_2]_s$, where C
 316 stands for carbon fiber and G for glass fiber. The evaluation of the objective function is made through a
 317 finite element analysis (FEA) software, ABAQUS®.

318 The design variables are the angles of the fiber orientation of 5 plies of the symmetric stacking
 319 sequence $[C_3/G_2]_s$ indicated as:

$$320 \mathbf{x} = (x_1, x_2, x_3, x_4, x_5)^T \quad (4)$$

321 These angles are discrete variables with allowable variation of 5° in the domain

$$322 -85^\circ \leq x_i \leq 90^\circ, i = 1, \dots, 5 \quad (5)$$

323 In the optimization process the default parameters of GLODS are used and the local minimums are
 324 found (after 10000 evaluations of the objective function) for each experimental setup: (a) plate fixed in
 325 one side, (b) plate fixed in two sides. The optimization carried out considered in the ABAQUS®
 326 simulations the theoretical fixed displacement and rotation as boundary conditions (i.e. totally fixed).
 327 These optimal results are presented in Table 10 for both experimental setups. The Hashin's failure criteria
 328 were also applied to the three optimized stacking sequences. For the optimized plates, the highest value
 329 for the Hashin's fiber criterion is 0.0027 for specimen P6 and for the Hashin's matrix criterion is 0.0148
 330 for specimen P8. Both values are bellow one, which means that no failure is expected in the optimized
 331 plates for the tested load.

332

333 **Table 10.** Optimal solutions obtained by GLODS for each experimental setup, with two local minima for the case of plate fixed in
 334 two sides.

Specimen	Experimental test	Number of plies	Material stacking sequence	Ply fiber orientations	Displacement value [mm]
P6	One side fixed			[15/15/15/0/15] _s	2.130
P7	Two side fixed	10	$[C_3/G_2]_s$	[-60/90/-60/60/-45] _s	0.358
P8	Two side fixed			[-60/-60/75/-60/-60] _s	0.357

335

3366. **EXPERIMENTAL VALIDATION OF THE OPTIMIZED LAMINATES**

337 Once the optimized stacking sequence of ply fiber orientations is obtained, it is important to validate
 338 the optimization process by performing experimental tests on the proposed optimized solutions. For
 339 experimental validation, the three optimized specimens configurations (P6, P7 and P8 of Table 10)
 340 obtained in the optimization process were manufactured using the same technique as the previous
 341 specimens P1, P2, P3, P4 and P5.

342 Table 11 presents the experimental and numeric results of the out of plane displacement at the
 343 location of the applied load, including the numerical results for both cases of totally and partially fixed
 344 rotation boundary conditions.

345

346 **Table 11.** Out of plane displacements results of the optimized specimens P6, P7 and P8.

Specimen	Experimental displacement [mm]	Computational displacement – totally fixed [mm]	Computational displacement – partially fixed [mm]
P6	2.35	2.130	2.420
P7	0.60	0.358	0.576
P8	0.62	0.357	0.582

347

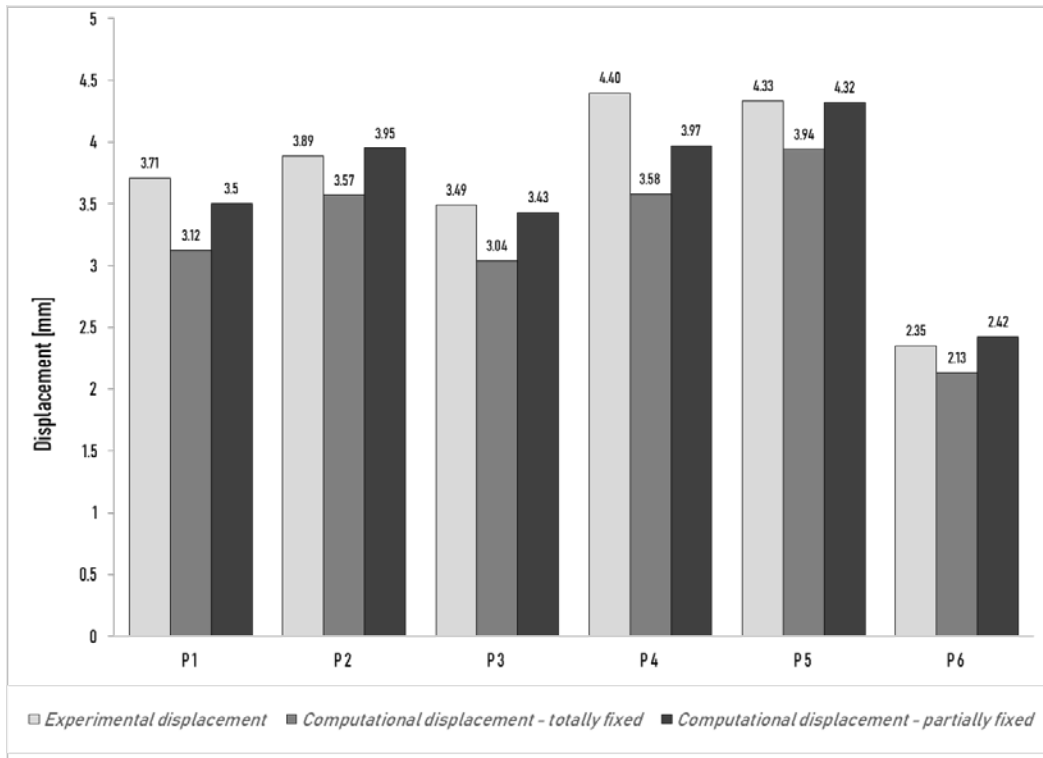
348 Figures 7 and 8 compare the out of plane displacement results for the non-optimized specimens (P1,
 349 P2, P3, P4 and P5) against the optimized specimens (P6, P7 and P8) considering the one side fixed
 350 condition and the two sides fixed condition, respectively.

351 It is quite clear from Figures 7 and 8 the significant decrease of the out of plane displacement of the
 352 optimized specimens found by GLODS, P6, P7 and P8, when compared to the non-optimized specimens,
 353 P1, P2, P3, P4 and P5, for both sets of boundary conditions. This decrease in out-of-plane displacement
 354 from non-optimized laminates to optimized laminates is in accordance with previous optimization works
 355 using GLODS with 100% carbon fiber reinforced laminates under combined loading bending and torsion
 356 [26].

357

358

359



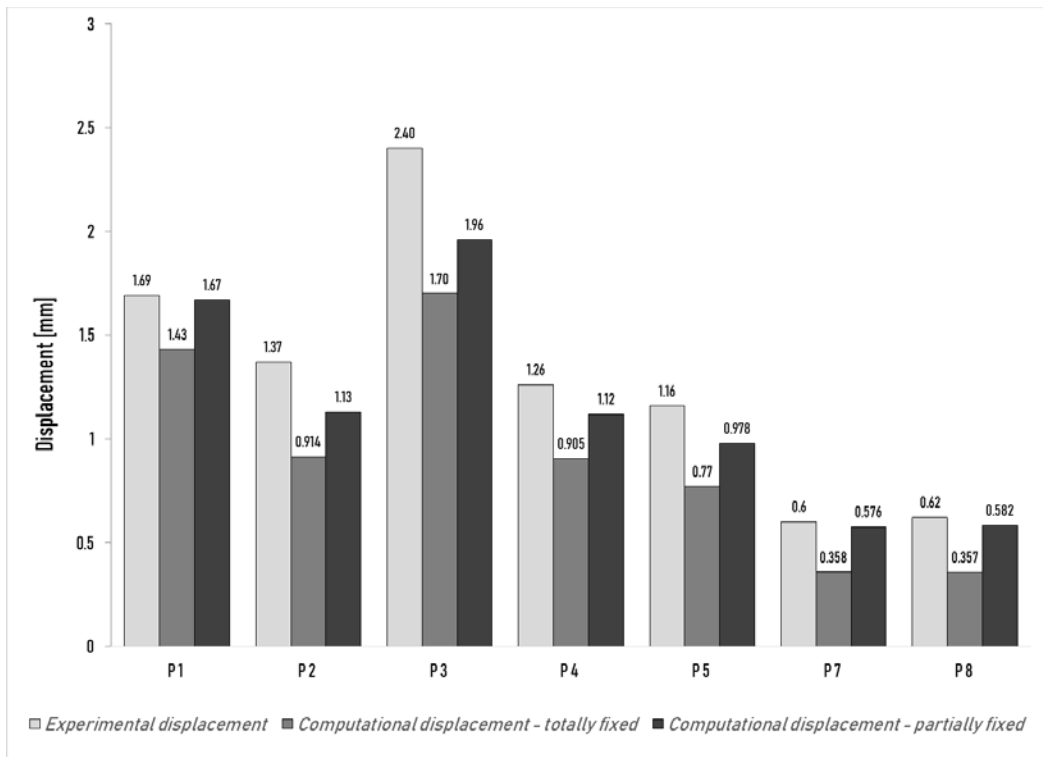
360

361

362

363

Figure 7. Comparison of experimental and computational values obtained for the non-optimized specimens (P1, P2, P3, P4 and P5) and the optimized specimens (P6) considering the plate fixed in one side.



364

365

366

367

Figure 8. Comparison of experimental and computational values obtained for the non-optimized specimens (P1, P2, P3, P4 and P5) and the optimized specimens (P7 and P8) considering the plate fixed in two sides.

368 It is important to recognize that experimental measurements are considered to be the true deformation
369 of the composite specimens, although in reality experimental errors may be inherently present.
370 Specifically, the manufacturing process of each sample may introduce a few unforeseeable imperfections
371 [26, 33], since the laminate quality in Hand Lay-Up is very dependent on the skills of the producer, just as
372 the curing process and the final cutting of the specimen. Meanwhile, the thickness of each carbon fiber
373 ply and each glass fiber ply was calculated with mean values of measurements of the carbon fiber and
374 glass fiber laminates, respectively. Each of the five specimens were measured in 9 points and it was
375 verified that the estimated thickness of the carbon fiber and glass fiber plies had a very good
376 approximation. However, the fact that these values were taken into account in the development of the
377 finite element models may also have contributed to some deviations in the computational results.
378 Additionally, the calculation of the elastic properties of the carbon fiber and glass fiber plies was done by
379 a numerical method based on a homogenization theory. This method depends on the definition of
380 geometric parameters of the Representative Elementary Volume, and as such, it is possible that small
381 errors are associated with these properties relatively to the ply material properties of the specimens tested
382 in the experiments.

383 The inherent uncertainties altogether, namely, the specimens manufacture using Hand Lay-Up, the
384 real ply material properties and the complexity of replicating numerically the exact boundary conditions
385 placed in the experimental setup tests, naturally introduce some small expected discrepancy between the
386 experimental and computational results. Even so, the optimization process is shown to be quite
387 successful. Not only the computational results of the optimized specimens are validated by the
388 experimental tests (within a reasonable discrepancy), but also the desired improvement of the
389 performance of the hybrid composites plates is achieved in more than 30% when compared to some non-
390 optimized plates. Furthermore, an important aspect that is demonstrated by the optimization of these
391 hybrid carbon-glass epoxy composites is that the most common ply fiber orientations of 0° , 90° and even
392 $\pm 45^\circ$ adopted for various laminated composites applications rather limits somewhat their performance. In
393 a real combined loading condition of bending and torsion, different ply fiber orientations should be
394 explored in the interest of maximizing the hybrid laminated composites performance. This has also been
395 observed for single fiber laminates [26].

396

3977. CONCLUSIONS

398 The increasing relevance of hybrid carbon-glass epoxy composites, whose mechanical properties can
399 be tailored to take advantage of the better properties of both fiber types, relies on their potential for an
400 increased performance compared to standard composites using one type of fiber alone. The aim of this
401 work was to optimize the hybrid composite stacking sequence, using the ply fiber orientation as design
402 variable to minimize the maximum out of plane displacement under combined loading of bending and
403 torsion. In order to study normal operation conditions, a load within the material linear elastic domain and
404 prior to any damage initiation was considered, as verified throughout the experimental tests and the
405 results from the Hashin's failure criteria.

406 The optimization process is shown to be quite successful, contributing to improve the performance of
407 the hybrid composites plates in more than 30% when compared to some non-optimized plates. In
408 addition, it is shown that ply fiber orientations different to the most usual 0° , 90° and $\pm 45^\circ$ should be
409 explored in the interest of maximizing the hybrid laminated composites performance under combine
410 loading of bending and torsion.

411

412 CONFLICT OF INTEREST

413 None.

414

415 ACKNOWLEDGEMENTS

416 This work was supported by "Fundação para a Ciência e a Tecnologia" (FCT), through the Institute of
417 Mechanical Engineering (IDMEC) under the Associated Laboratory for Energy, Transports and
418 Aeronautics (LAETA), Project UID/EMS/50022/2013, and by the Netherlands Organisation for Scientific
419 Research (NWO), project number 14366.

420

421 REFERENCES

422 [1] Pandya KS, Veerraju C, Naik NK. Hybrid composites made of carbon and glass woven fabrics under
423 quasi-static loading. *Materials and Design* 2011; 32(7): 4094-4099.

- 424 [2] Enfedaque A, Molina-Aldareguía JM, Gálvez F, González C, Llorca J. Effect of glass fiber
425 hybridization on the behavior under impact of woven carbon fiber/epoxy laminates. *Journal of Composite*
426 *Materials* 2010; 44(25): 3051-3068.
- 427 [3] Dong C, Ranaweera-Jayawardena HA, Davies IJ., Flexural properties of hybrid composites reinforced
428 by S-2 glass and T700S carbon fibres. *Composites Part B: Engineering* 2012; 43(2): 573-581.
- 429 [4] Dong C, Davies IJ. Flexural strength of bidirectional hybrid epoxy composites reinforced by E glass
430 and T700S carbon fibres. *Composites Part B: Engineering* 2015; 72: 65-71.
- 431 [5] Kretsis G. A review of the tensile, compressive, flexural and shear properties of hybrid fibre-
432 reinforced plastics. *Composites* 1987; 18(1): 13-23.
- 433 [6] Naik NK, Ramasimha R, Arya H, Prabhu SV, ShamaRao N. Impact response and damage tolerance
434 characteristics of glass-carbon/epoxy hybrid composite plates. *Composites Part B: Engineering* 2001;
435 32(7): 565-574.
- 436 [7] Hosur MV, Adbullah M, Jeelani S. Studies on the low-velocity impact response of woven hybrid
437 composites. *Composite Structures* 2005; 67(3): 253-262.
- 438 [8] Sayer M, Bektaş NB, Demir E, Çallioğlu H. The effect of temperatures on hybrid composite laminates
439 under impact loading. *Composites Part B: Engineering* 2012; 43(5): 2152-2160.
- 440 [9] González EV, Maimí P, Sainz de Aja JR, Cruz P, Camanho PP. Effects of interply hybridization on
441 the damage resistance and tolerance of composite laminates. *Composite Structures* 2014; 108(1): 319-
442 331.
- 443 [10] Zhang J, Chaisombat K, He S, Wang CH. Hybrid composite laminates reinforced with glass/carbon
444 woven fabrics for lightweight load bearing structures. *Materials and Design* 2012; 36: 75-80.
- 445 [11] Dong C, Davies IJ. Optimal design for the flexural behaviour of glass and carbon fibre reinforced
446 polymer hybrid composites. *Materials and Design* 2012; 37: 450-457.
- 447 [12] Dong C, Davies IJ. Flexural and tensile strengths of unidirectional hybrid epoxy composites
448 reinforced by S-2 glass and T700S carbon fibre. *Materials and Design* 2014; 54: 955-966.
- 449 [13] Kalantari M, Dong C, Davies IJ. Multi-objective analysis for optimal and robust design of
450 unidirectional glass/carbon fibre reinforced hybrid epoxy composites under flexural loading. *Composites*
451 *Part B: Engineering* 2016; 84: 130-139.

- 452 [14] Kalantari M, Dong C, Davies IJ. Multi-objective robust optimisation of unidirectional carbon/glass
453 fibre reinforced hybrid composites under flexural loading. *Composite Structures* 2016; 138: 264-275.
- 454 [15] Swolfs Y, Gorbatiikh L, Verpoest I. Fibre hybridisation in polymer composites: A review.
455 *Composites Part A* 2014;, 67: 181–200.
- 456 [16] Gurdal Z, Haftka RT, Hajela P. *Design and Optimization of Laminated Composite Materials*. Wiley-
457 Interscience, New York, 1999.
- 458 [17] Todoroki A, Haftka RT. Stacking sequence optimization by a genetic algorithm with a new recessive
459 gene like repair strategy. *Composites Part B* 1998; 29(3): 277–285.
- 460 [18] Toropov VV Jones R, Willment, Funnell M. Weight and manufacturability optimization of
461 composite aircraft components based on a genetic algorithm. 6th World Congress of Structural and
462 Multidisciplinary Optimization, Rio de Janeiro, Brazil, 2005.
- 463 [19] Zhu X, He R, Lu X, Ling X, Zhu L, Liu B. A optimization technique for the composite strut using
464 genetic algorithms. *Materials & Design (1980-2015)* 2015; 65: 482–488.
- 465 [20] Ghiasi H, Pasini D, Lessard L. Optimum stacking sequence design of composite materials part I:
466 constant stiffness design. *Composite Structures* 2009;. 90(1): 1–11.
- 467 [21] Custodio AL, Madeira JFA, Vaz AIF, Vicente LN. Direct Multisearch for Multiobjective
468 Optimization. *SIAM Journal on Optimization* 2011; 21(3): 1109–1140.
- 469 [22] Araújo AL, Madeira JFA, Mota Soares CM, Mota Soares CA. Optimal design for active damping in
470 sandwich structures using the Direct MultiSearch Method. *Composite Structures* 2013; 105: 29–34.
- 471 [23] Madeira JFA, Araújo AL, Mota Soares CM, Mota Soares CA, Ferreira AJM. Multiobjective design
472 of viscoelastic laminated composite sandwich panels. *Composites Part B: Engineering* 2015; 77: 391–
473 401.
- 474 [24] Madeira JFA, Araújo AL, Mota Soares CM, Mota Soares CA. Multiobjective optimization of
475 viscoelastic laminated sandwich structures using the Direct MultiSearch Method. *Composite Structures*
476 2015; 147: 229–235.
- 477 [25] Franco Correia VM, Madeira JA, Araújo AL, Mota Soares CM. Multiobjective design optimization
478 of laminated composite plates with piezoelectric layers. *Composite Structures* 2017; 169: 10–20.

- 479 [26] Monte SMC, Infante V, Madeira JFA, Moleiro F. Optimization of fibers orientation in a composite
480 specimen. *Mechanics of Advanced Materials and Structures* 2017; 24(5): 410-416.
- 481 [27] Melro AR, Camanho PP, Pinho ST. Influence of geometrical parameters on the elastic response of
482 unidirectional composite materials. *Composite Structures* 2012; 94(11): 3223–3231.
- 483 [28] Guedes J, Kikuchi N. Preprocessing and postprocessing for materials based on the homogenization
484 method with adaptive finite element methods. *Computer Methods in Applied Mechanics and Engineering*
485 1990; 83(2) 143–198.
- 486 [29] Coelho PG, Amiano LD, Guedes JM, Rodrigues HC. Scale-size effects analysis of optimal periodic
487 material microstructures designed by the inverse homogenization method. *Computers & Structures* 2015;
488 174: 21-32.
- 489 [30] Hashin Z. Failure Criteria for Unidirectional Fiber Composites. *Journal of Applied Mechanics* 1980;
490 47: 329–334.
- 491 [31] Kress G. Examination of Hashin's failure criteria for Part B of the second world wide failure
492 exercise: comparison with test data. *Journal of Composite Materials* 2013; 47, (6): 7867–7891.
- 493 [32] Custodio AL, Madeira JFA. GLODS: Global and Local Optimization using Direct Search. *Journal of*
494 *Global Optimization* 2015; 62 (1): 1-28.
- 495 [33] Hucker MJ, Bond IP, Haq S, Bleay S, Foreman A. Influence of manufacturing parameters on the
496 tensile strengths of hollow and solid glass fibres. *Journal of Materials Science* 2002; 37(2): 309–315.